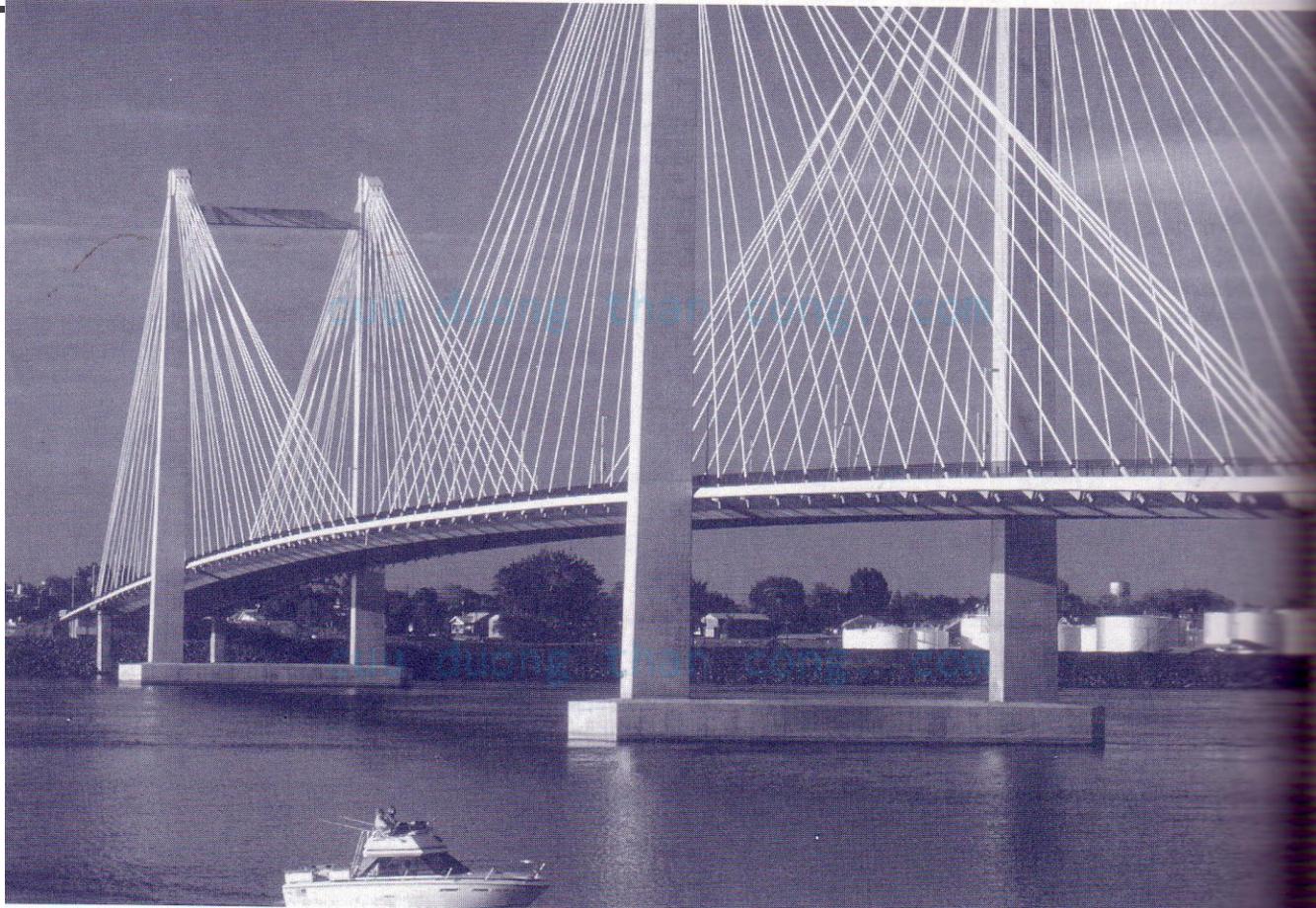


CHAPTER 2: STRESS AND STRAIN CONCEPTS

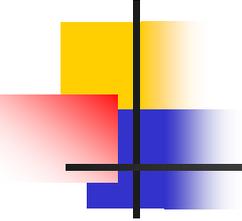
ASSOC.PROF BÙI CÔNG THÀNH
FACULTY OF CIVIL ENGINEERING

CHAPTER 2: STRESS AND STRAIN CONCEPTS



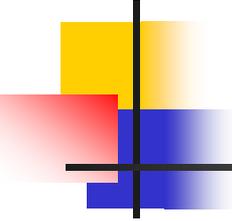
Associate Prof. Dr. Bui Cong Thanh

Faculty of Civil Engineering



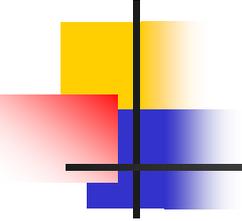
CONTENTS (NỘI DUNG)

- Introduction (Giới thiệu chung)
- Concept of Stress (Khái niệm σ/s)
- Uniaxial Problems (Bài toán 1 chiều)
- Shear Stress – Strain (τ/s – b/d cắt)
- Stresses on Inclined Sections
- Stress – Strain Diagrams
- Safety Factor – Allowable Stresses



I/ INTRODUCTION (GIỚI THIỆU)

- This chapter → Mechanics of Materials (Strength of Materials or Mechanics of Solids) cuu duong than cong. com
- **Mechanics of Materials** is a branch of Mechanics that develops relationships between **the external loads** applied to a **deformable body** (vật thể biến dạng được) and **the internal forces** (nội lực) acting within the body



I/ INTRODUCTION (GIỚI THIỆU) (continued)

- Mechanics of materials → stresses, strains and deflections based on 3 fundamental principles:

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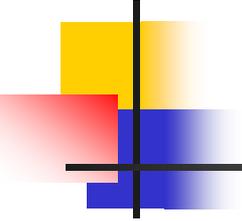
✓ Statics: Equilibrium of forces (stresses)

✓ Geometry: Compatibility

deformations:

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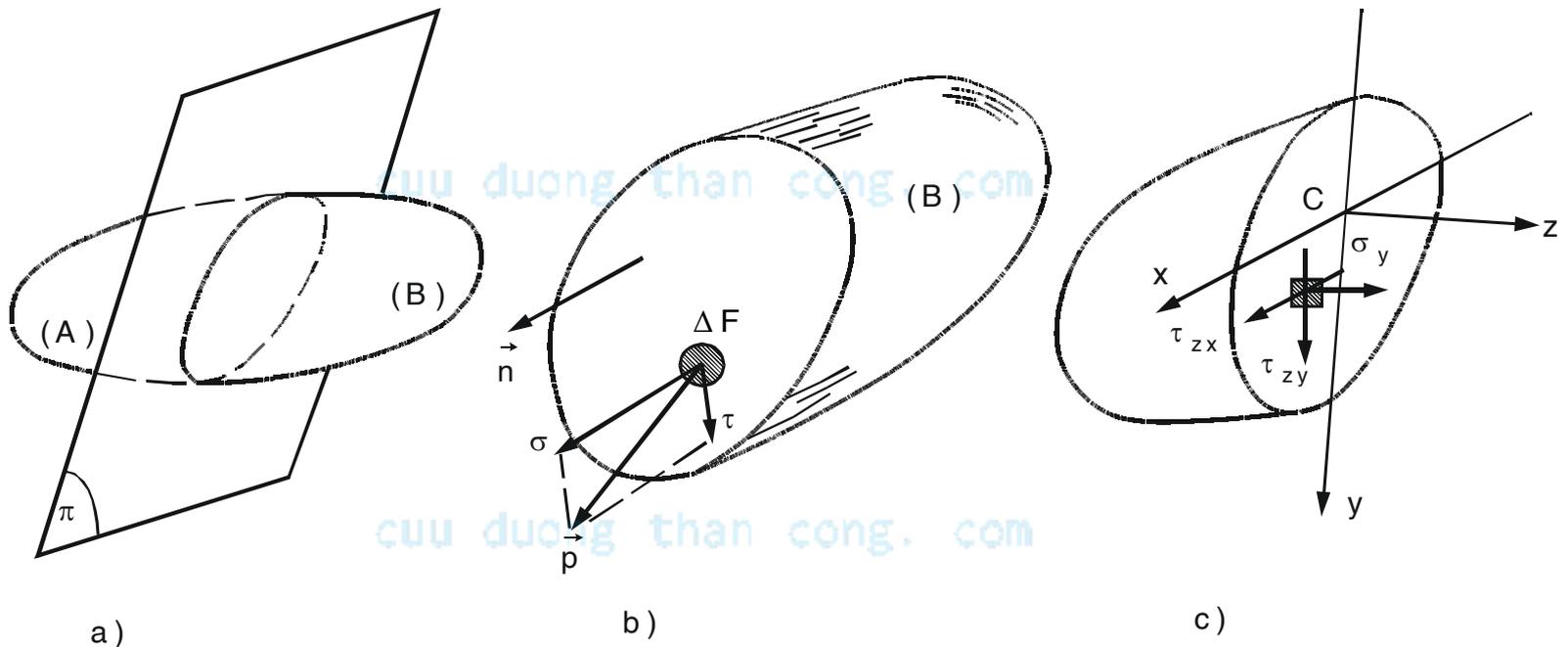
✓ Constitutive Laws; stress-strain relations



I/ INTRODUCTION (GIỚI THIỆU) (continued)

- **Boundary conditions: (điều kiện biên)**
 - ✓ Static ones (điều kiện biên tĩnh học)
 - ✓ Kinematical ones (đ/k biên động học)
- **Material Hypotheses (Giả thiết v/l):**
 - ✓ Perfect Elasticity
 - ✓ Continuity, Homogeneity, Isotropy

II/ CONCEPT OF STRESS (KHÁI NIỆM VỀ ỨNG SUẤT)



A body in equilibrium (Vật thể cân bằng)

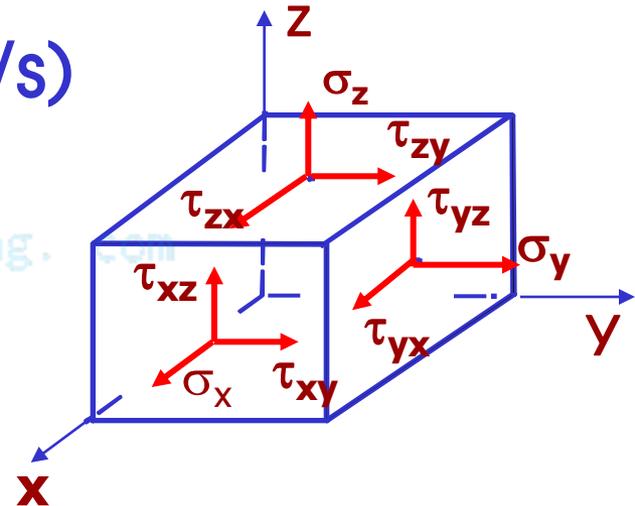
II/ CONCEPT OF STRESS (KHÁI NIỆM VỀ ỨNG SUẤT)

- Stresses at a point (Ứng suất tại 1 điểm)

$$\vec{p} = \lim_{\Delta F \rightarrow 0} \frac{\vec{\Delta P}}{\Delta F} \quad \left\{ \begin{array}{l} \sigma : \text{normal stress} \\ \tau : \text{shearing stress} \end{array} \right.$$

- Stress tensor (tenxơ ứng suất)

$$\sigma = \begin{pmatrix} \sigma_x & \tau_{xy} & \tau_{xz} \\ \tau_{yx} & \sigma_y & \tau_{yz} \\ \tau_{zx} & \tau_{zy} & \sigma_z \end{pmatrix}$$



II/ CONCEPT OF STRESS (KHÁI NIỆM VỀ ỨNG SUẤT)

- Sign convention of normal stresses:

✓ $\sigma_x > 0$: tensile stress

✓ $\sigma_x < 0$: compression stress

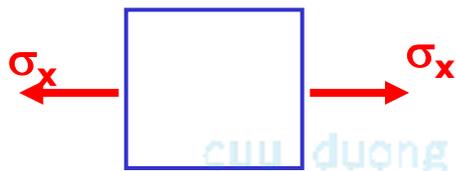
- From the equil. eqs. of moments about the x-, y-, & z-axes, one has:

$$\tau_{yz} = \tau_{zy}; \tau_{xz} = \tau_{zx}; \tau_{xy} = \tau_{yx}$$

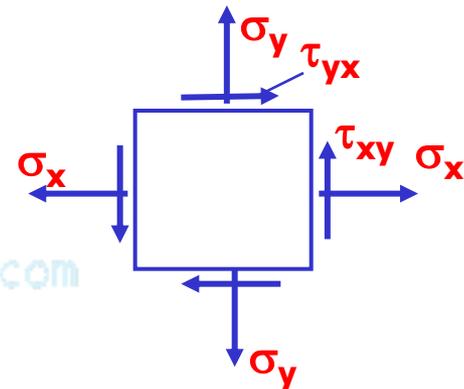
→ 6 independent stress components

II/ CONCEPT OF STRESS (KHÁI NIỆM VỀ ỨNG SUẤT)

- Uniaxial Stress State (Trạng thái ư/s đơn)
- State of plane stresses / two dimensional stress state. (Trạng thái ứng suất phẳng)

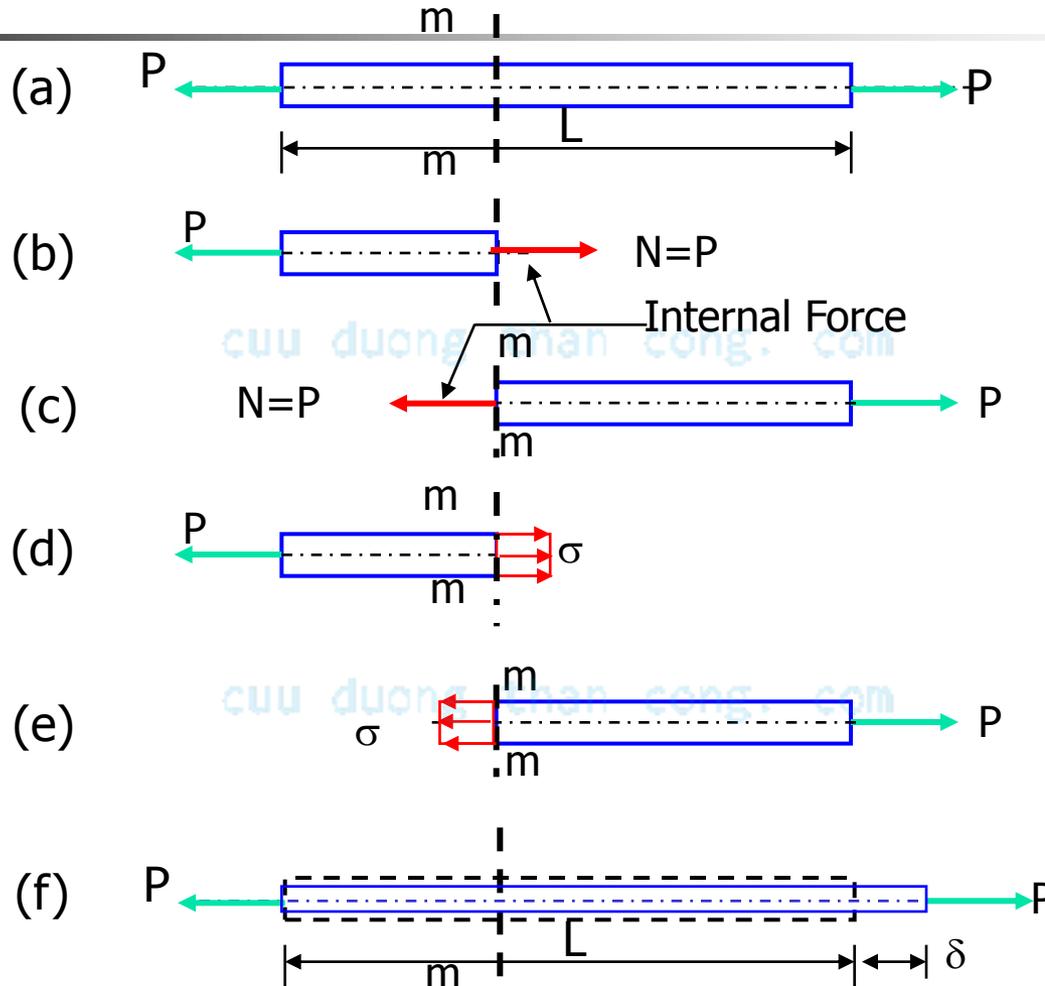


Uniaxial Stress
State (TUS đơn)



Plane Stress State
(TUS phẳng)

III/UNIAXIAL PROBLEMS (BÀI TOÁN ĐƠN TRỤC)



III/UNIAXIAL PROBLEMS (BÀI TOÁN ĐƠN TRỤC)

- Internal force: From a cut section → $N = P$ represents also the **resultant of normal, tensile stress**
- Normal stress: *at a section* far from the bar ends, the normal stress can be regarded **uniform**, or considered as average value

$$\sigma = \frac{N}{A} = \frac{P}{A}$$

III/UNIAXIAL PROBLEMS (BÀI TOÁN ĐƠN TRỤC)

- Deformation (Biến dạng):

- ✓ Longitudinal strain (Biến dạng dài):

$$\varepsilon = \frac{\delta}{L}$$

- ✓ Lateral strain (Biến dạng ngang):

$$\varepsilon_{\text{lateral}} = -\nu \varepsilon$$

ν - Poisson's ratio, determined by experiment

III/UNIAXIAL PROBLEMS (BÀI TOÁN ĐƠN TRỤC)

- Dilatation = unit volume change (Độ dẫn thể tích):

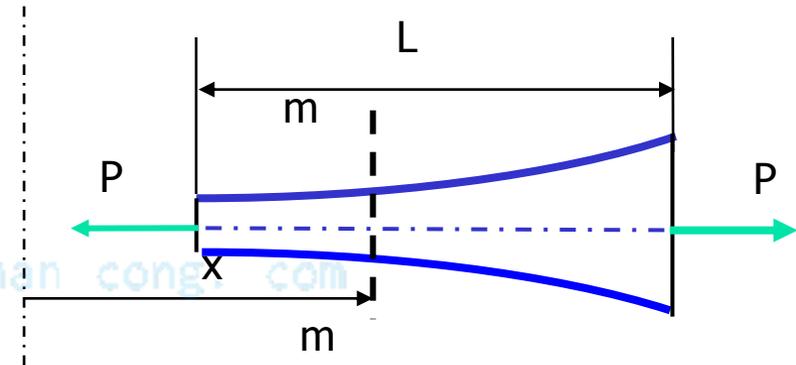
$$\theta = \frac{\Delta V}{V} = \varepsilon (1 - 2\nu)$$

(ν cannot pass the value 0,5, since any larger value would mean that the volume of a bar decreases when it is being stretched!)

III/UNIAXIAL PROBLEMS (BÀI TOÁN ĐƠN TRỤC)

- Axially Loaded Straight Bar of Varying Cross Section (Thanh có tiết diện thay đổi chịu tải dọc trục)

$$\delta = \int_0^L \varepsilon(x) dx$$



IV/ SHEARING STRESS & STRAIN (ỨNG SUẤT – BIẾN DẠNG CẮT)

- **Shear stress** = stress component acting in the plane of considered section

In many engineering problems, **shear stress τ** is assumed as **uniformly distributed** over certain sections for simplicity

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$$\tau_{\text{aver}} = \frac{V}{A}$$

IV/ SHEARING STRESS & STRAIN (ỨNG SUẤT – BIẾN DẠNG CẮT)

- State of pure shear (Trạng thái trượt thuần túy)

An element being acted upon by shear stress only is in *state of pure shear*

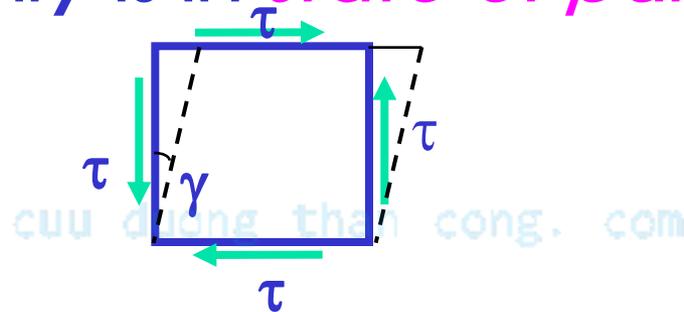
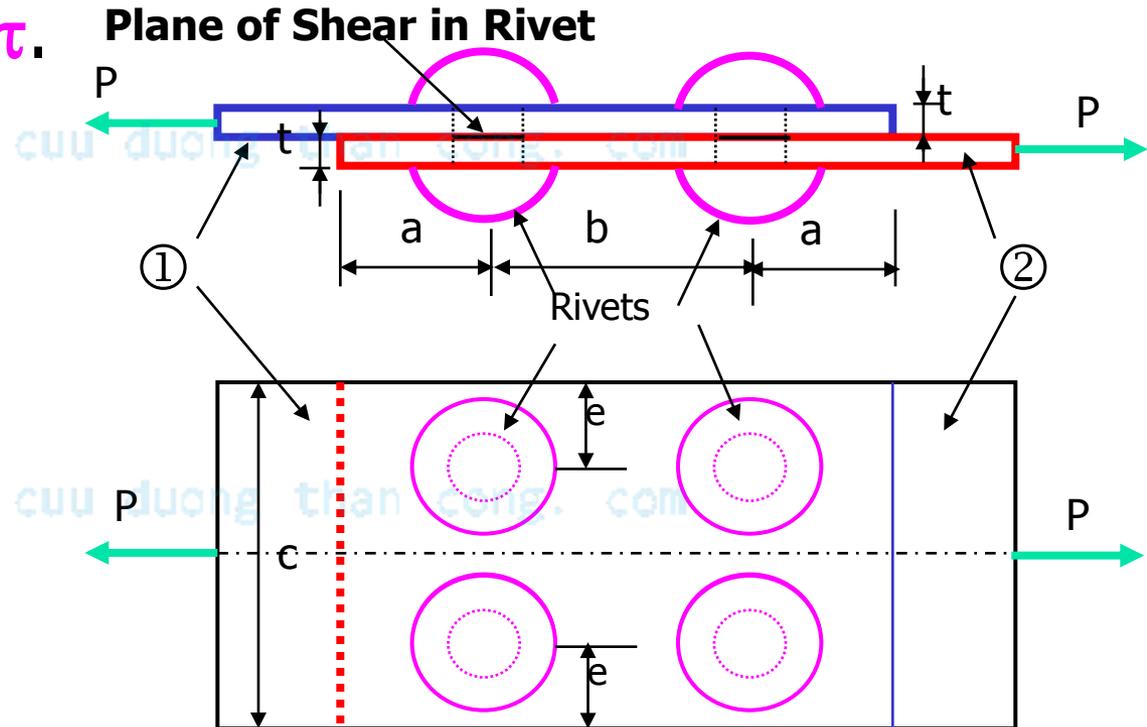


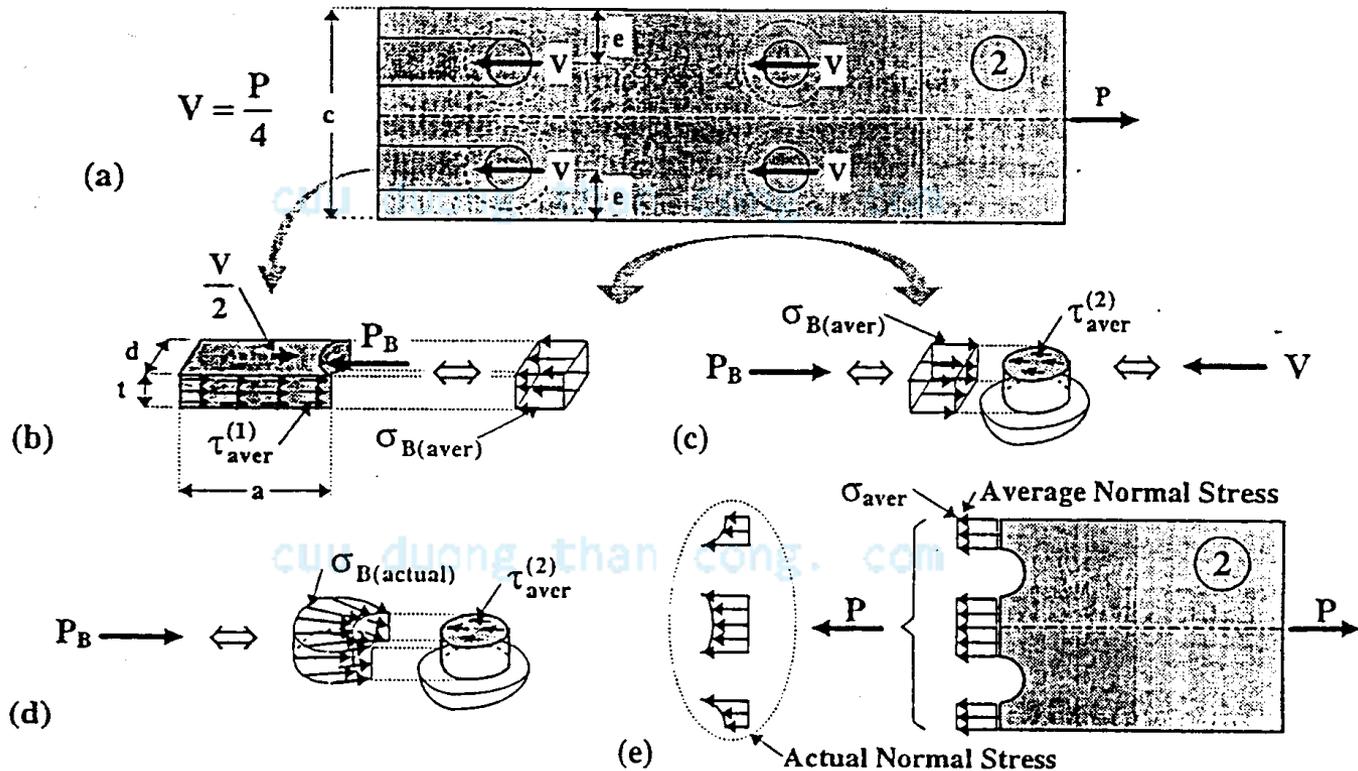
Fig.2.7 State of pure shear

IV/ SHEARING STRESS & STRAIN (ỨNG SUẤT – BIẾN DẠNG CẮT)

- Shear Strain, γ : the change in right angle due to τ .



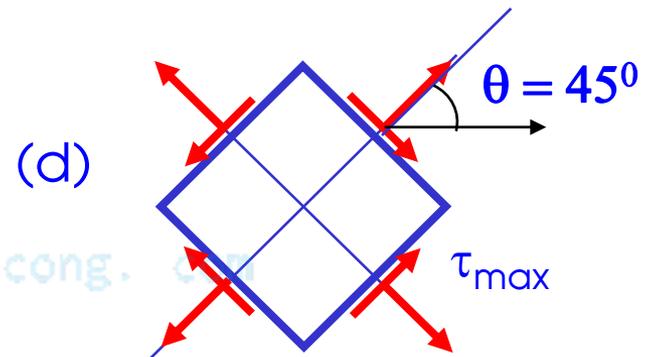
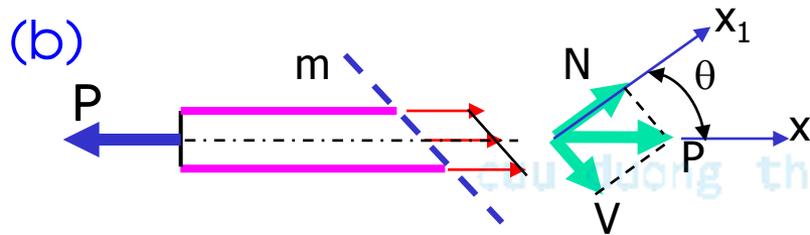
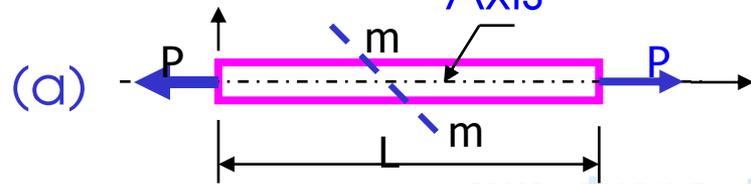
IV/ SHEARING STRESS & STRAIN (ỨNG SUẤT - BIẾN DẠNG CẮT)

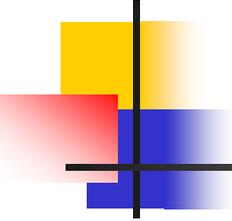


V/ STRESSES ON INCLINED SECTIONS (U/s trên m/c nghiêng)

Centroidal

Axis

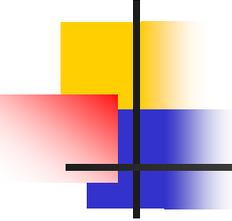




V/ STRESSES ON INCLINED SECTIONS (U'/s trên m/c nghiêng)

- An axially loaded bar is cut by a section m-m inclined with respect to the longitudinal axis, and whose normal makes an angle of θ with the x-axis (Fig.a).
- **We try to find σ_{θ} ? τ_{θ} ?**

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V/ STRESSES ON INCLINED SECTIONS (U's trên m/c nghiêng)

- Hypotheses :

- ✓ The section m-m is relative far from the bar ends
- ✓ On m-m the stress is uniformly distributed in x-direction, whose resultant is equal to P

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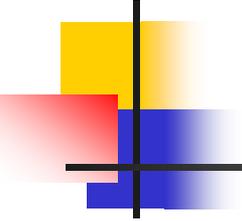
V/ STRESSES ON INCLINED SECTIONS (U/s trên m/c nghiêng)

- The normal & shear stresses:

$$\sigma_{\theta} = \frac{N}{A_{\theta}} = \sigma_x \cos^2 \theta \quad \tau_{\theta} = -\frac{V}{A_{\theta}} = -\frac{\sigma_x}{2} \sin 2\theta$$

- Maximal shear stress & corres. norm. str.

$$\tau_{max} = \left| \tau_{\theta} \Big|_{\theta=45^{\circ}} \right| = \frac{\sigma_x}{2} \quad \sigma_{\theta} \Big|_{\theta=45^{\circ}} = \frac{\sigma_x}{2}$$



VI/ STRESS-STRAIN DIAGRAMS

- Elastic and Linear-Elastic Material Behavior
 - ✓ If removing the applied loads restores the original shape and size → elastic material
 - ✓ A linear stress-strain relationship → linearly elastic material

VI/ STRESS-STRAIN DIAGRAMS (continued)

- Hooke's Law

✓ The linear stress-strain relationships

$$\sigma = E \varepsilon \qquad \tau = G \gamma$$

E - Modulus of Elasticity or Young's Modulus

G – Shear Modulus of Elasticity

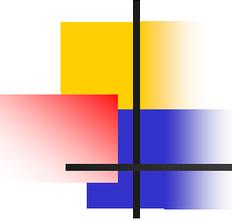
Unit of E, G: Pa, GPa = 10⁹ Pa

VI/ STRESS-STRAIN DIAGRAMS (continued)

- Relation among E , G and ν in Isotropic Materials (determined from theory and experiments)

$$G = \frac{E}{2(1 + \nu)}$$

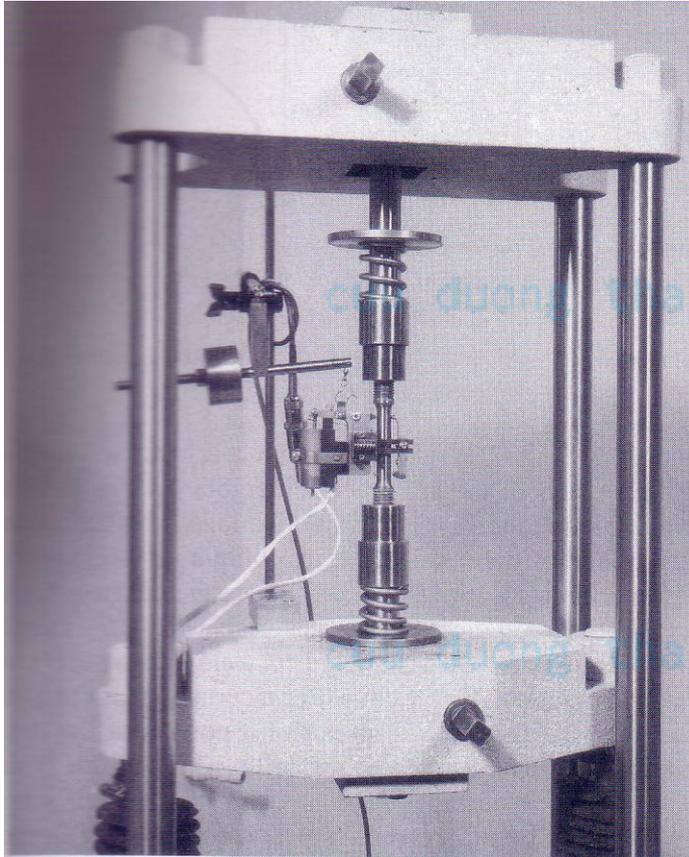
⇒ “The materials elastic properties E , G & ν are not independent”



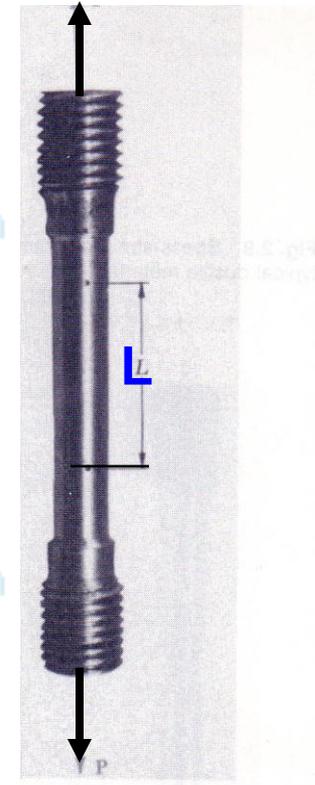
VI/ STRESS-STRAIN DIAGRAMS (continued)

- Stress-Strain Diagrams. Tensile Test of Mild Steel
 - ✓ Generalities:
 - $(\sigma-\varepsilon)$ diagrams → basic mechanical properties of structural materials.
 - From tensile & compression tests
 - Category of Materials: ductile and brittle (fragile) materials

VI/ STRESS-STRAIN DIAGRAMS (continued)



Máy vạn năng

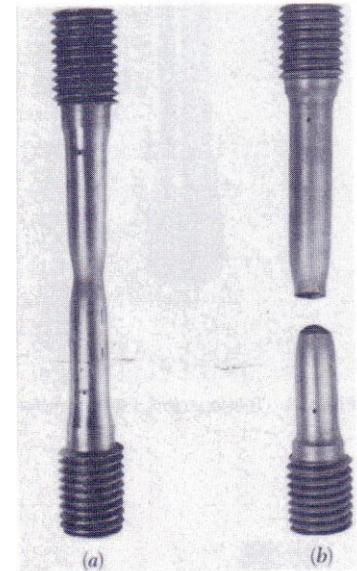
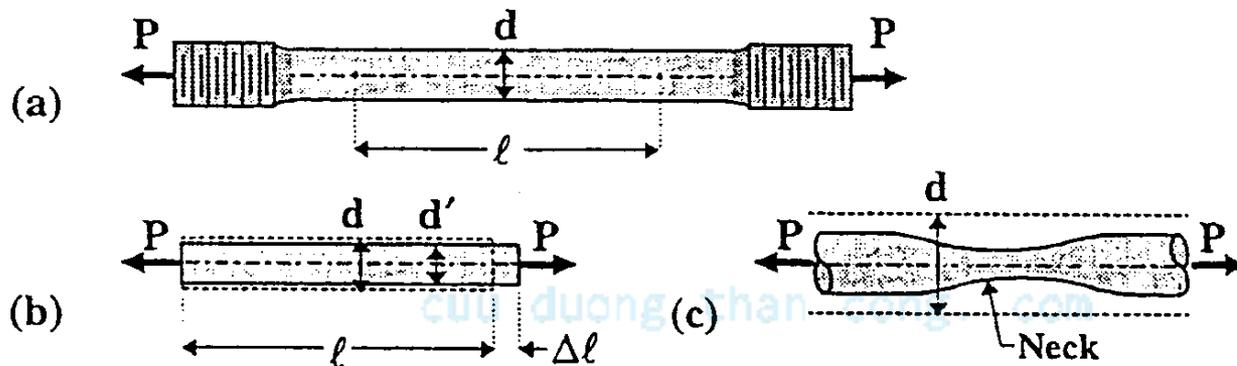


Mẫu thử kéo

VI/HOOKE'S LAW. STRESS-STRAIN DIAGRAMS (continued)

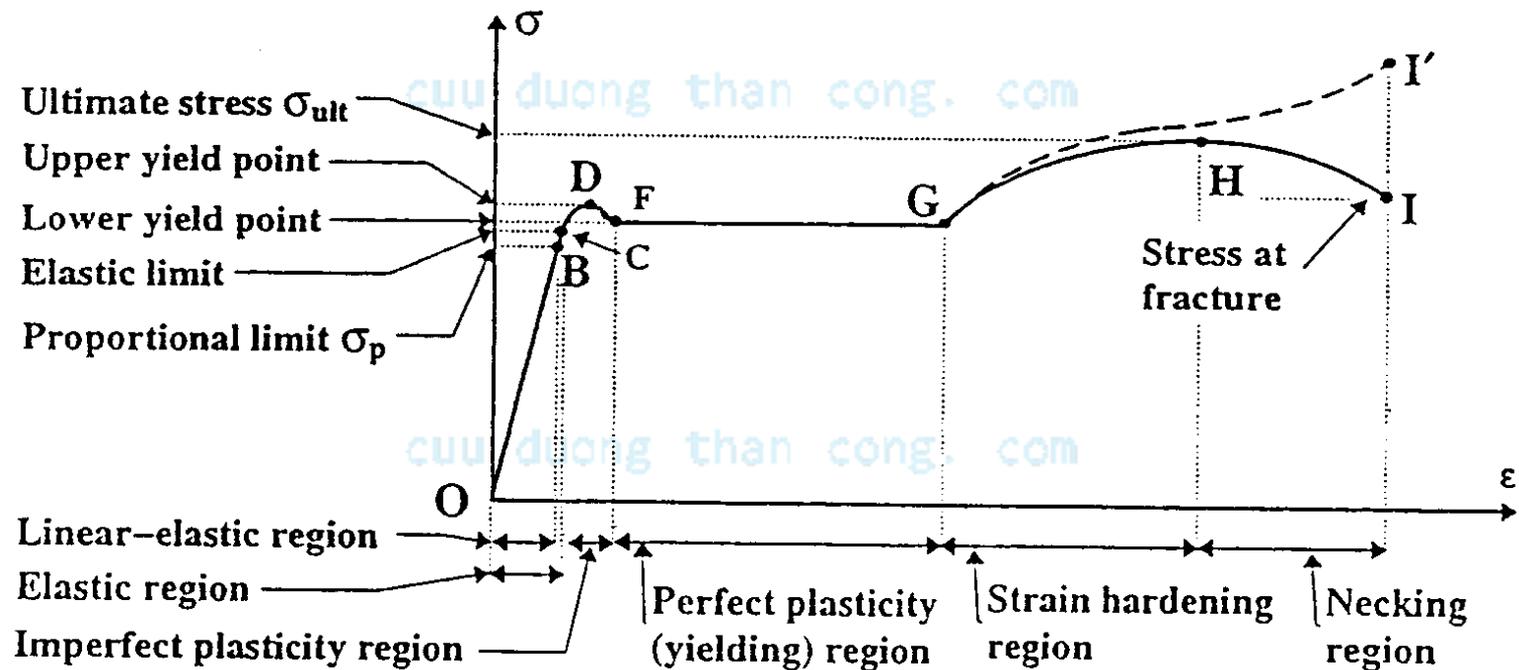
✓ Tensile Test Specimen of Mild Steel: a slender circular cross section bar:

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VI/ STRESS-STRAIN DIAGRAMS (continued)

✓ Real stress-strain diagram



VI/ STRESS-STRAIN DIAGRAMS (continued)

✓ Elastic domain: OB

- linear stress–strain relationship
- point B: **proportional limit**, σ_p
- Const slope: *Modulus of Elasticity*, E

✓ Transition domain: BCDF

- point C: **Elastic Limit** (B \equiv C)
- Point D: **the Upper Yield Point**,
- Point F: **the Lower Yield Point**,

$\sigma_y^{(u)}$
 $\sigma_y^{(l)}$

VI/ STRESS-STRAIN DIAGRAMS (continued)

✓ Region of Plasticity: FG

- Deformation develops under **constant yield stress**, $\sigma_y^{(u)} = \sigma_y^{(l)} = \sigma_y$

- G → end of the region of plasticity

✓ Strain-hardening & necking regions:

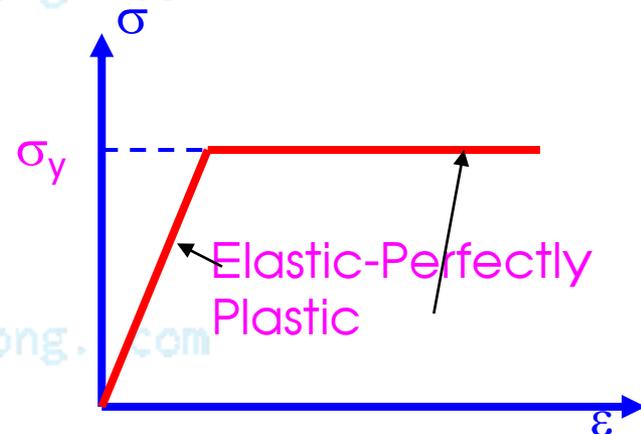
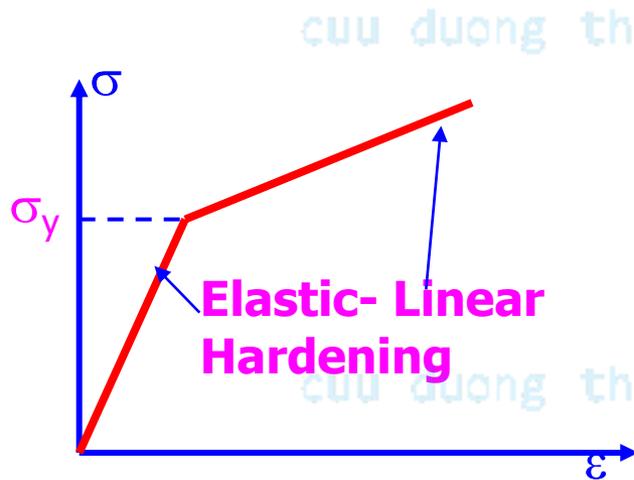
- Nonlinear Stress-Strain Relationship

- Point H: → **the ultimate stress**, σ_{ult}

- Point I: → **Fracture of the specimen**

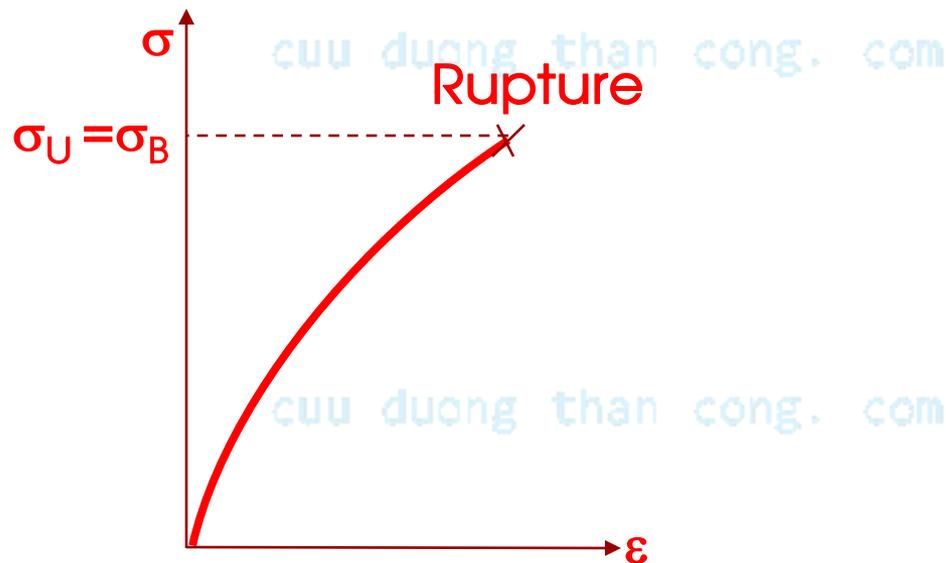
VI/ STRESS-STRAIN DIAGRAMS (continued)

- ✓ Idealised mild steel stress-strain diagrams

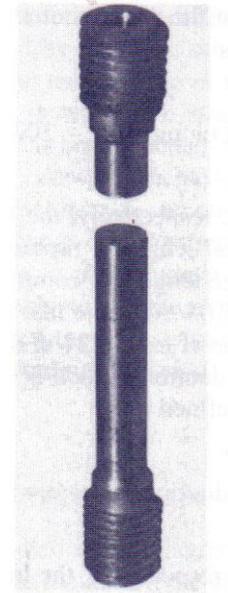


VI/ STRESS-STRAIN DIAGRAMS (continued)

✓ Tensile Test Specimen of brittle materials



Stress-strain diagram for a typical brittle material



Tested specimen of a brittle material

VII/ FACTOR OF SAFETY. ALLOWABLE STRESSES. SERVICE LOADS

- Factor of Safety:

Factor of safety $n = \frac{\text{actual strength}}{\text{required strength}} > 1$

- Allowable Stresses, σ_{allow} (or $[\sigma]$)

✓ Ductile Materials: $\sigma_{\text{allow}} = \frac{\sigma_y}{n}$, $n \approx 1.67$

For mild steel, $\sigma_y = 250\text{MPa} \Rightarrow (\sigma) = 150\text{MPa}$

$$\sigma_{\text{allow}} = \frac{\sigma_{\text{ult}}}{n}$$

✓ Brittle Materials: